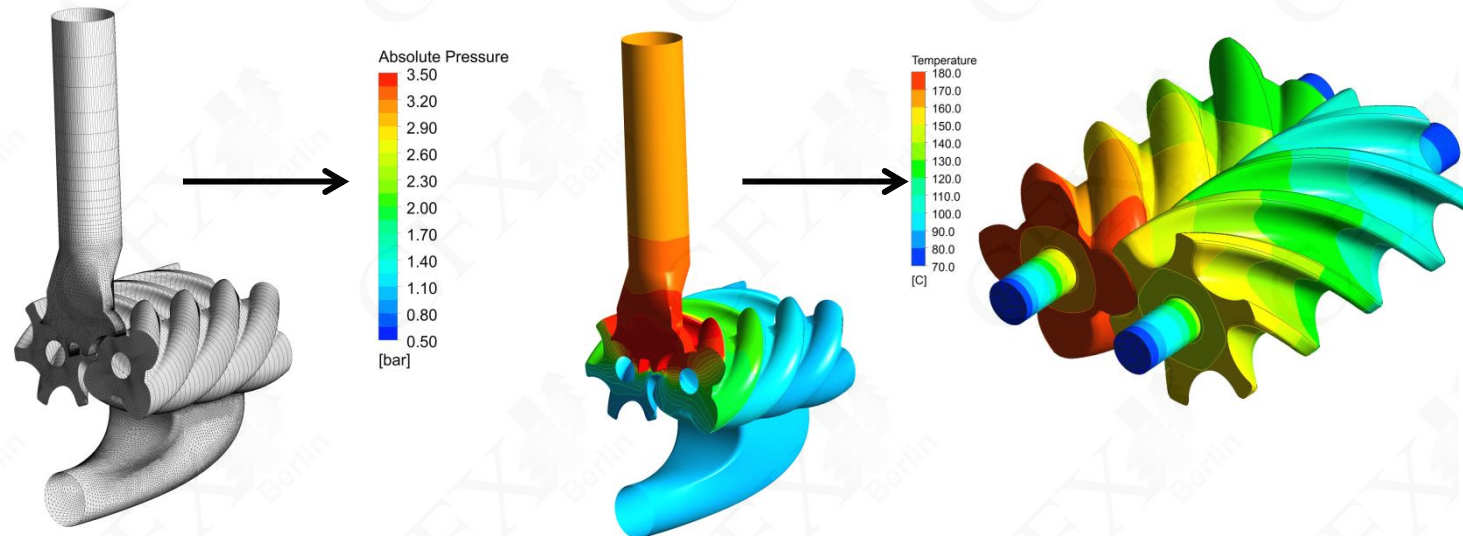


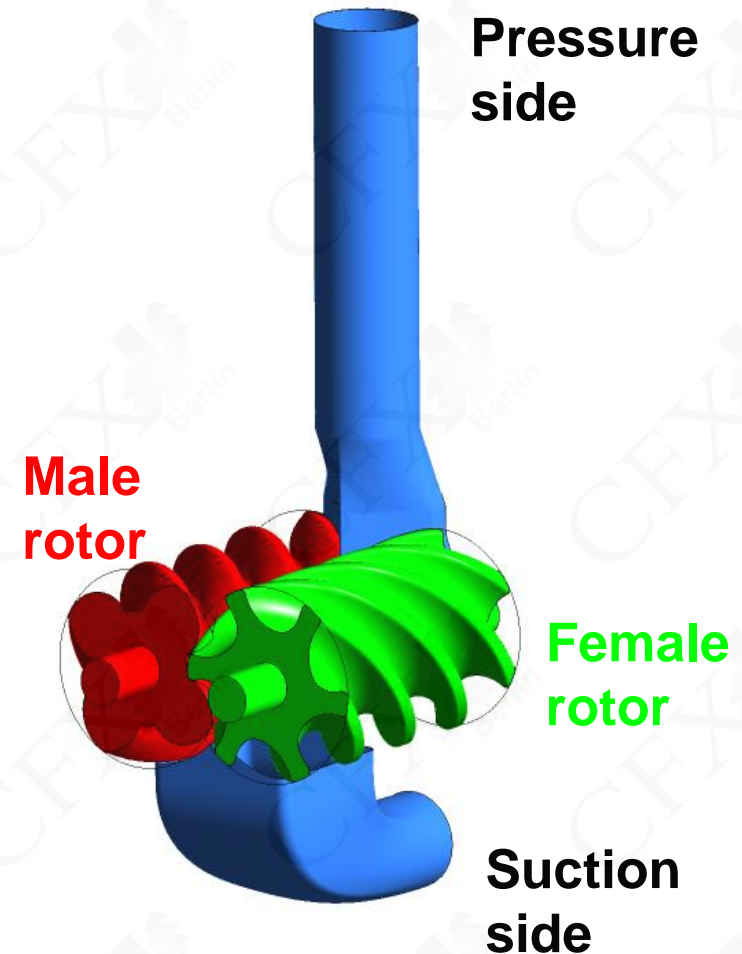
CFD Simulation of a Screw Compressor Including Leakage Flows and Rotor Heating

*9th International Conference on
Compressors and their Systems
London, 7th – 9th September 2015*

Dr. Andreas Spille-Kohoff
Jan Hesse
Ahmed El Shorbagy
CFX Berlin Software GmbH
Karl-Marx-Allee 90 A
10243 Berlin, Germany

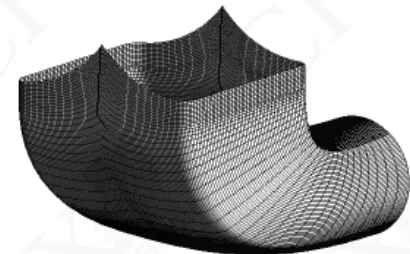
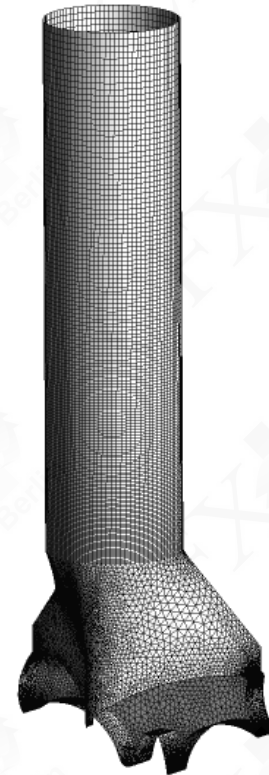
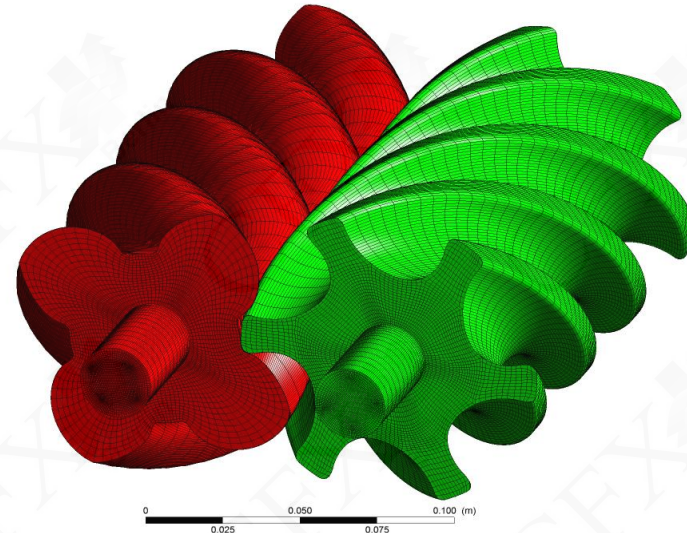


- Example case is from Master's Thesis of Ahmed El Shorbagy:
 - *“Auslegung, Konstruktion und numerische Simulation eines trockenlaufenden Schraubenverdichters und Vergleich der Simulationsergebnisse mit den Entwurfsanforderungen“*
Technical University Berlin, 2014
 - Design, construction, and numerical simulation of a dry-running screw compressor
 - Design: SRM profile 4+6 with rotor profile after „Schraubenverdichter“ by Lorenz Rinder (1979)
 - Construction: suction and pressure side with 3 different pressure ports
 - Simulation: setup, solution, and post-processing with ANSYS CFX
 - Extended by axial gaps and CHT in solids



Software and workflow:

- **Meshing**
 - TwinMesh for chamber meshes
 - ANSYS Meshing for stator and solid meshes
- **Pre-processing**
 - Session file from TwinMesh
 - ANSYS CFX-Pre
- **Solution**
 - ANSYS CFX Solver with User Fortran for reading of rotor meshes at run-time
 - ANSYS Structural for deformations
- **Post-Processing**
 - ANSYS CFD-Post



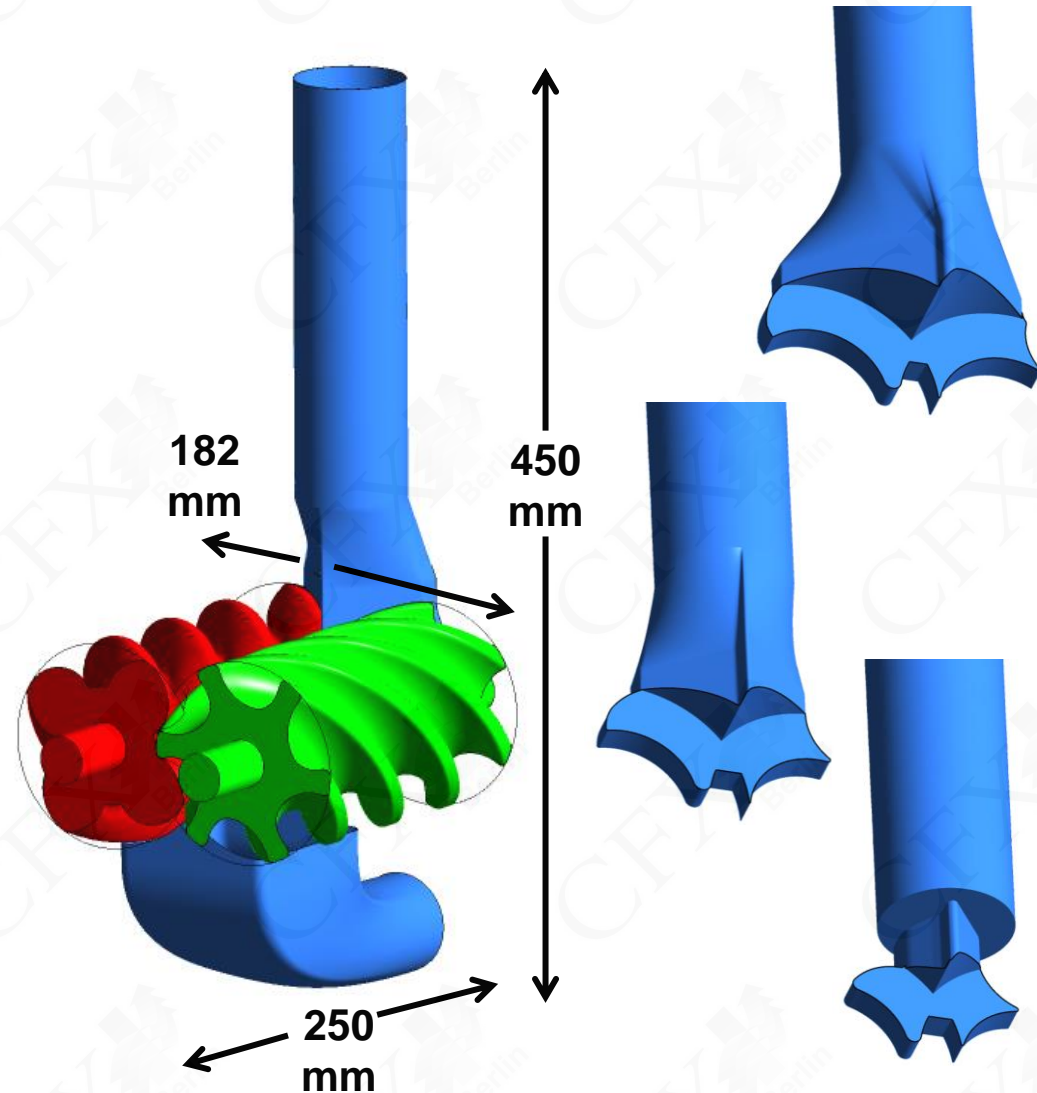
Geometry and dimensions:

- **Rotors:**

- Male: 4 lobes, \varnothing 102 mm, wrap angle 300°
- Female: 6 lobes, \varnothing 101.2 mm, wrap angle 200°
- Length: 168.3 mm
- Shafts: length 248.3 mm, \varnothing 24 mm
- Distance of rotation axes: 80 mm
- Clearances: radial 50 μm , interlobe 100 μm , axial 100 μm

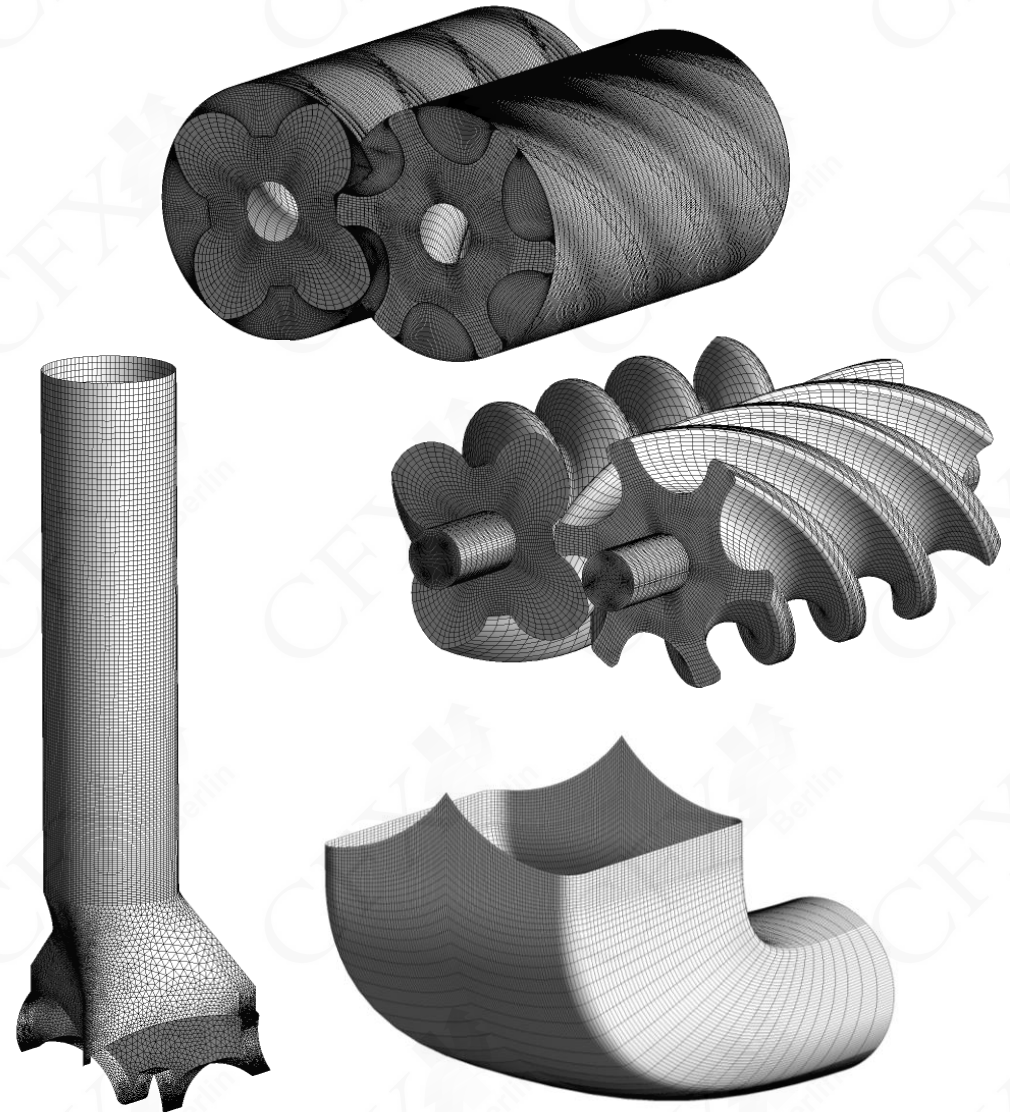
- **Ports:**

- Radial suction port
- Axial and radial pressure port in 3 variations for volume ratio 2.2, 2.7 and 3
- Ending in pipes with \varnothing 50-55 mm



Meshing:

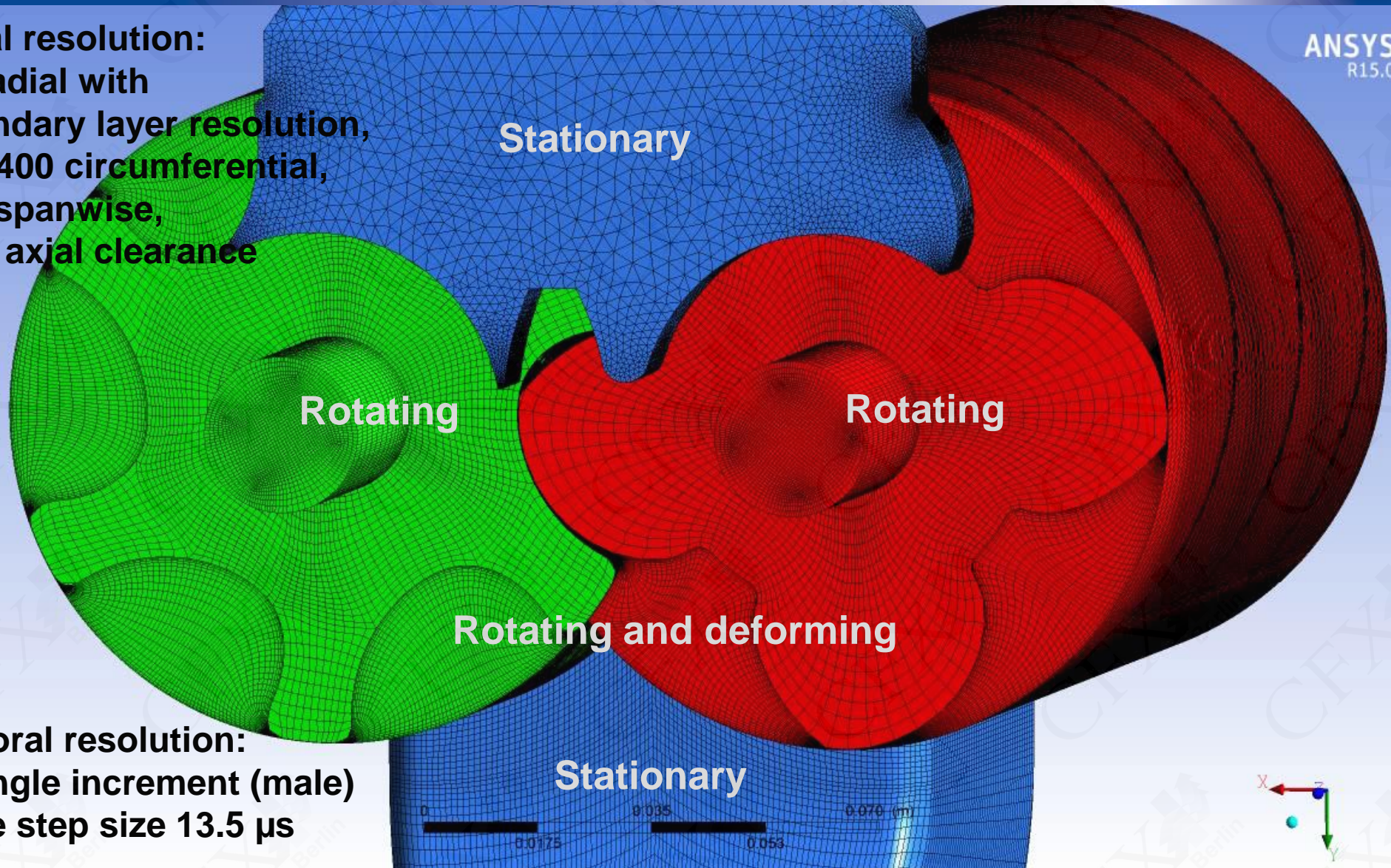
- **Rotors:**
 - Hexahedral meshes, each appr. 550 000 nodes, for rotating and deforming fluid regions around rotors including clearances from TwinMesh for each time step
 - Hexahedral meshes, each appr. 300 000 nodes, for solid rotors
- **Stator parts:**
 - Mixed mesh with hexahedrons, tetrahedrons and prisms (1 mio nodes)
- **Total mesh:**
 - 2.7 mio nodes
 - 3.5 mio elements



Meshing cont.

Spatial resolution:

- 20 radial with boundary layer resolution,
- 300-400 circumferential,
- 130 spanwise,
- 5-10 axial clearance



Temporal resolution:

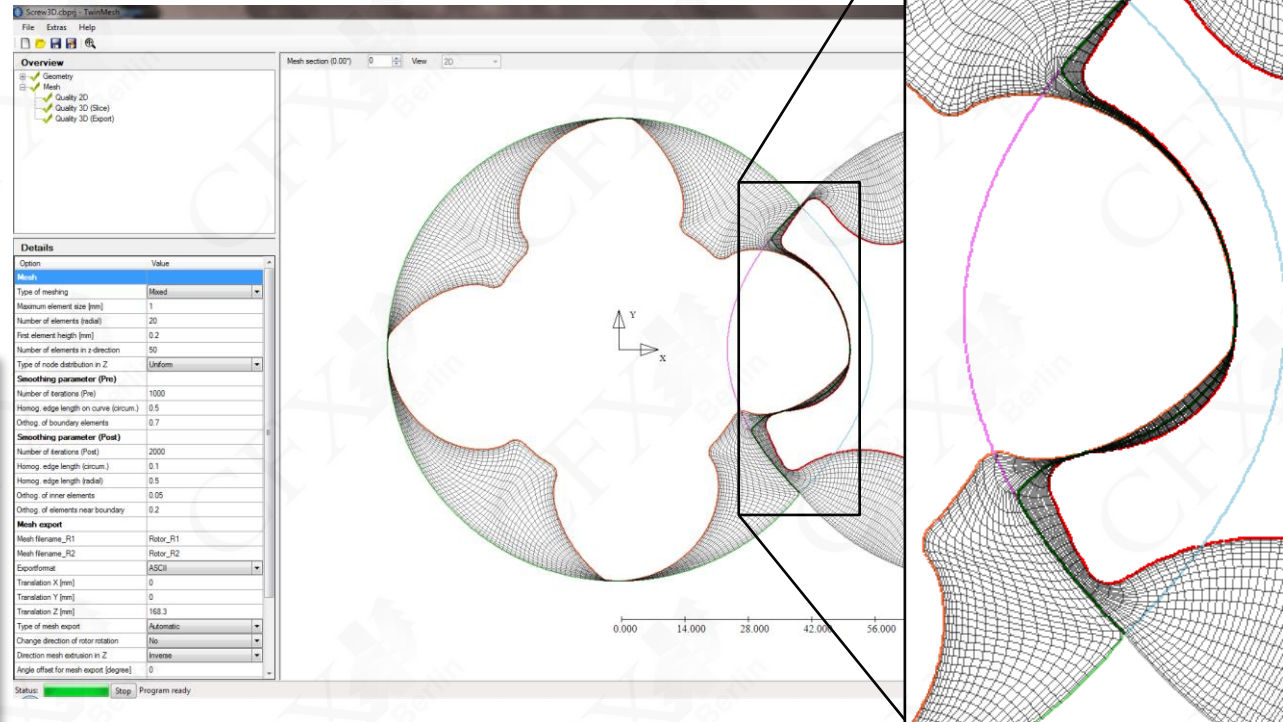
- 1° angle increment (male)
- Time step size 13.5 μ s

Meshing with TwinMesh



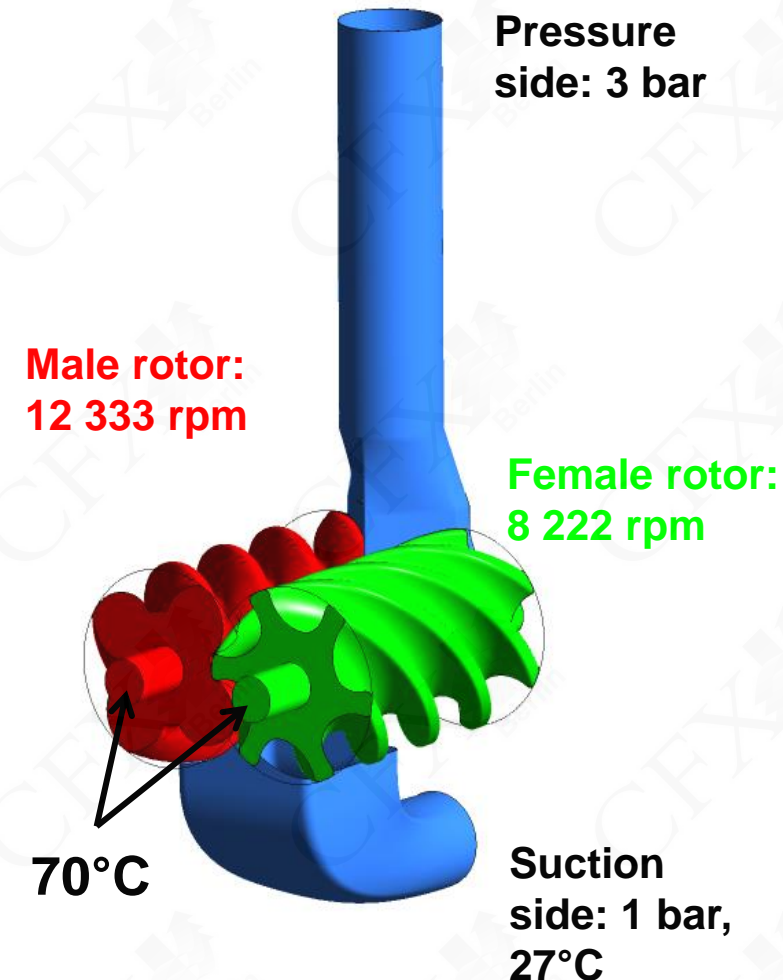
CAD
↓

- TwinMesh**
1. Import geometry
 2. Set boundary conditions
 3. Generate interfaces
 4. Apply mesh settings
 5. Check mesh quality
 6. Generate meshes
 7. Export meshes

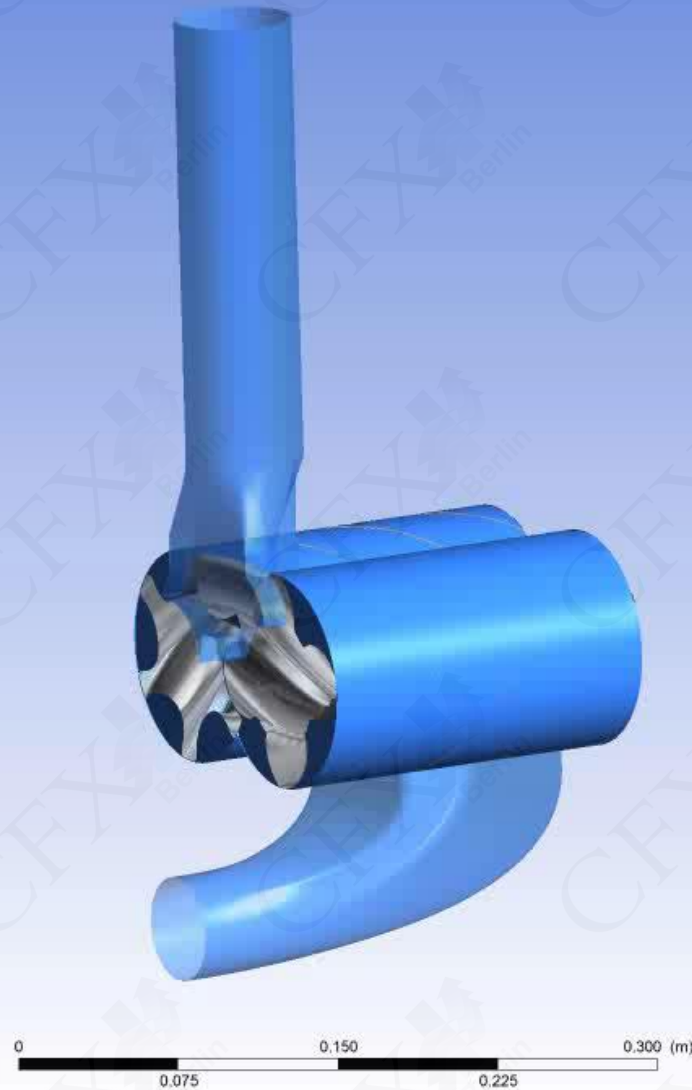


- Intuitive and comfortable GUI
- High quality structured meshes
- Gap sizes down to 1 µm
- Individual node distribution

- **Materials:**
 - Fluid: Air as Ideal Gas (dry compressor)
 - Solid: Steel
- **Models:**
 - SST turbulence model
 - Total energy model with viscous dissipation and CHT into solids
 - User Fortran for rotor meshes at run-time, mesh deformation for solid rotation
- **Boundary conditions:**
 - Rotation speed (male): 12 333 rpm
 - Suction side: opening at 1 bar and 27°C
 - Pressure side: opening at 3 bar
 - Non-reflecting boundary conditions
 - Adiabatic casing except shaft ends at 70°C
- **Solver:**
 - ANSYS CFX for 90 time steps of 1°
 - Restart with interpolation

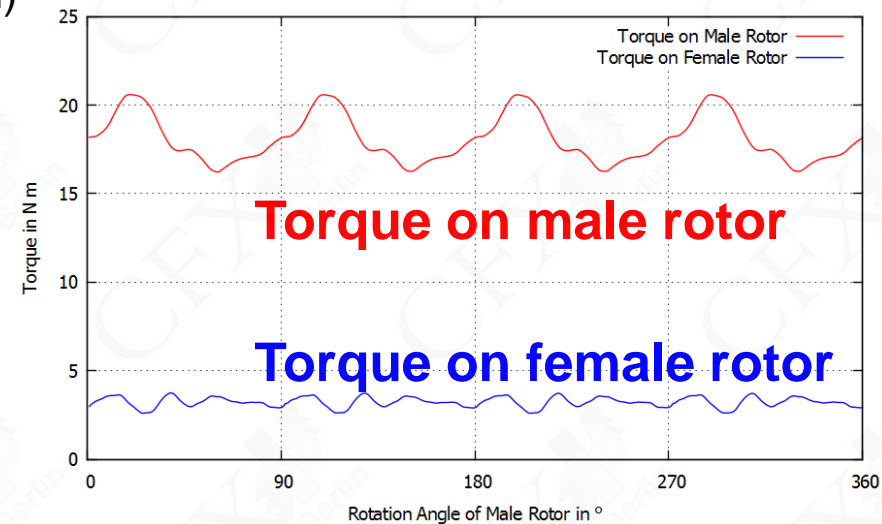
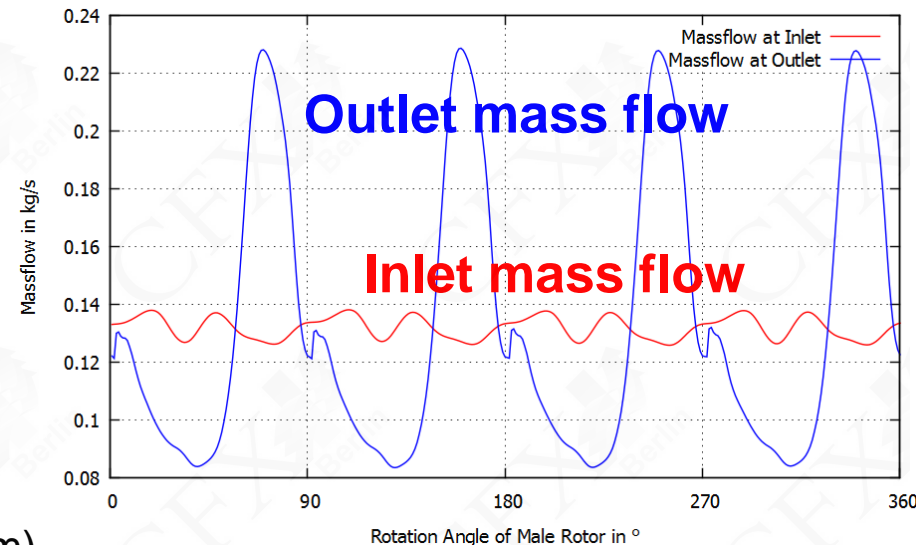


Result Overview

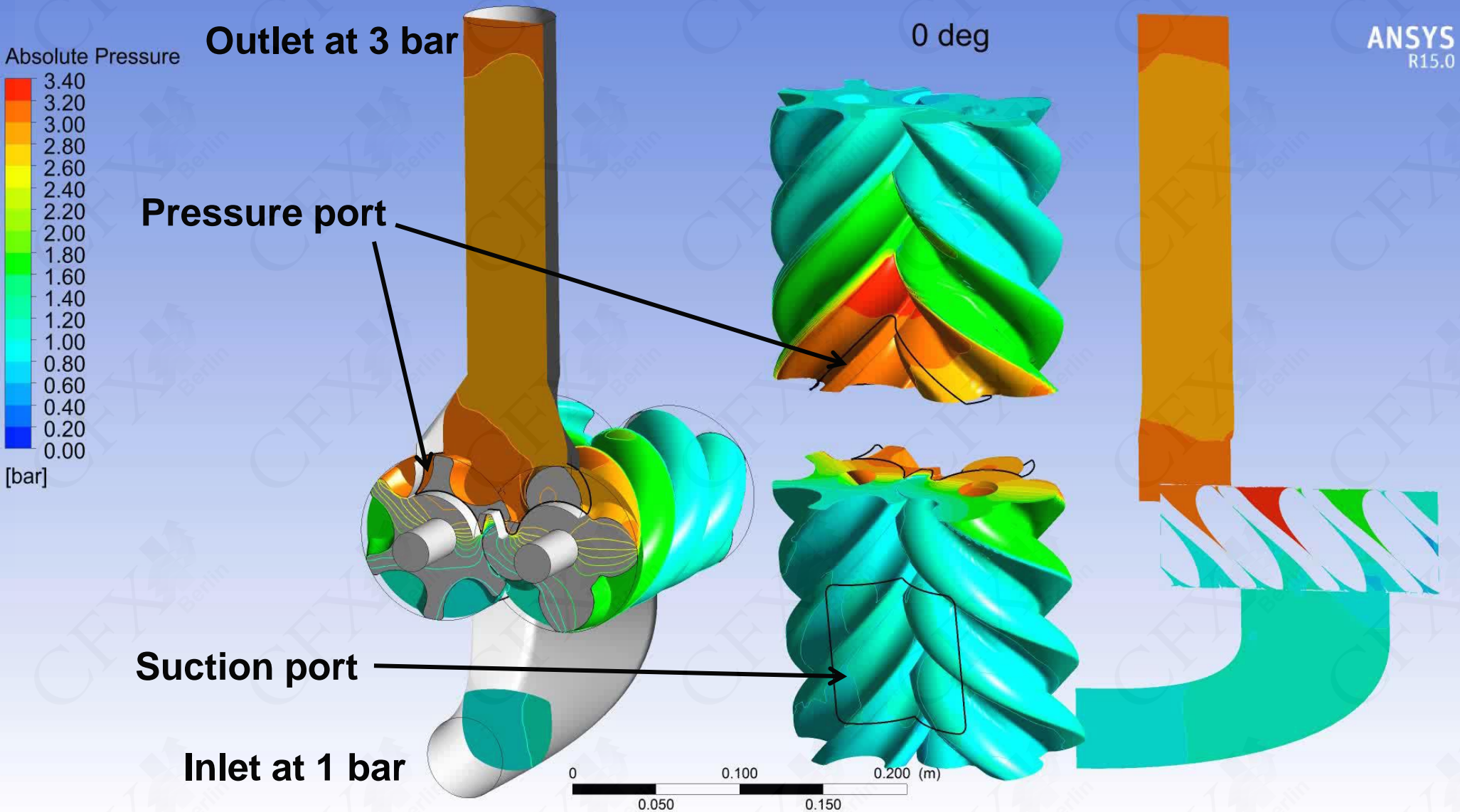


Quantitative results:

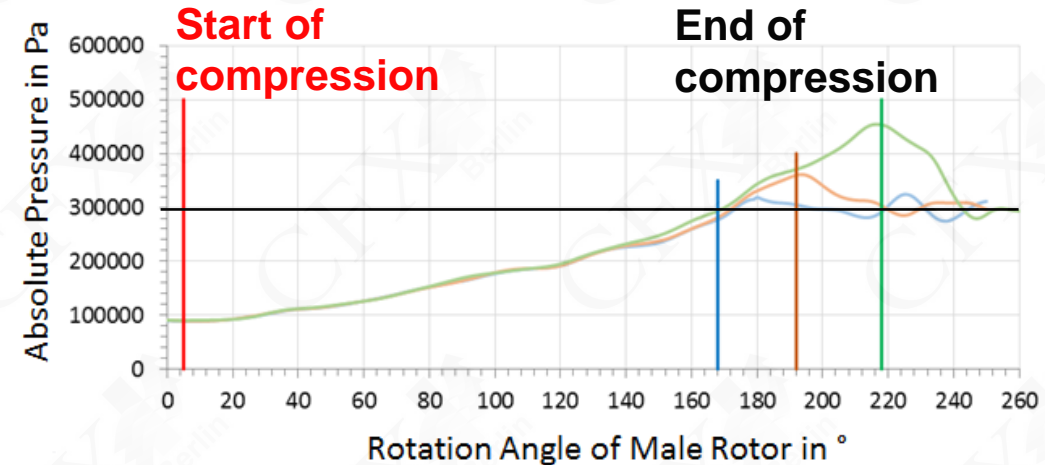
- **Mass flow:**
 - Average mass flow: 0.13 kg/s
 - Almost constant at inlet
 - High fluctuation at outlet
- **Torque:**
 - High, almost constant torque on male rotor (average 18.2 Nm)
 - Low torque on female rotor (average 3.2 Nm)
- **Power:**
 - Total power consumption 25.1 kW
- **Computation time:**
 - ANSYS CFX-15.0.7 with MeTiS partitioning
 - 9 hours for 90° on 8 cores Intel Xeon E5-2637 v2 with Platform MPI
 - 20 GB memory for double precision solver



Absolute Pressure



- Chamber pressure for different pressure ports:
 - Absolute pressure for monitor point in male rotor chamber over male rotor angle
 - Almost identical pressure increase during compression
 - Pressure port for estimated pressure ratio **3** shows undercompression
 - Pressure ports for estimated pressure ratios **4** and **4.6** show overcompression
 - Slight higher pressure during compression for **4.6** due to increased gap flow



Pressure port for pressure ratio

3

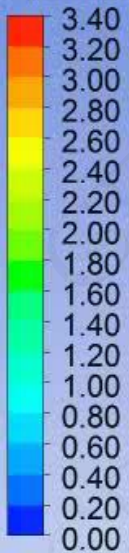
4

4.6



Flow through Axial and Radial Gaps

Absolute Pressure

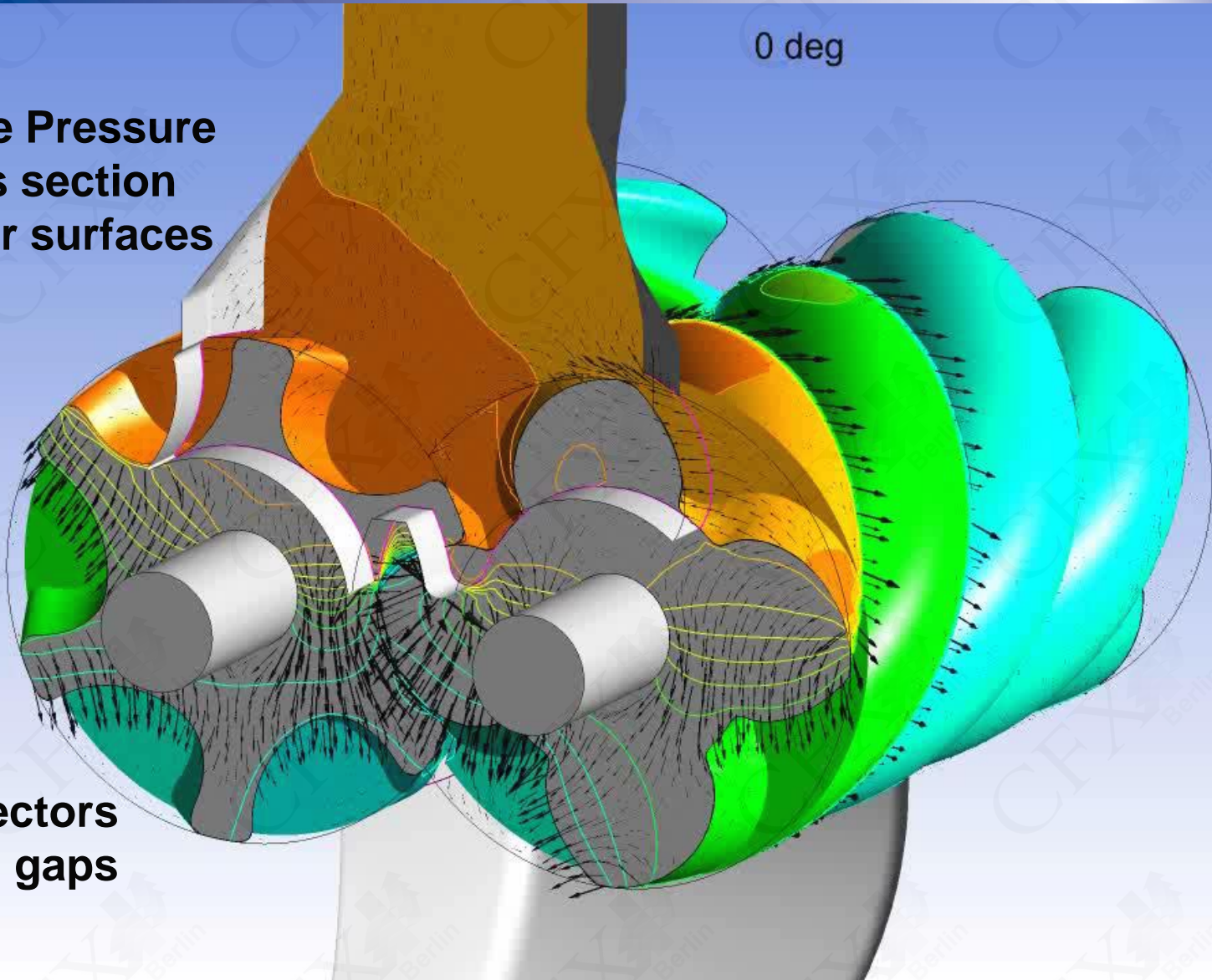


[bar]

**Absolute Pressure
on cross section
and rotor surfaces**

0 deg

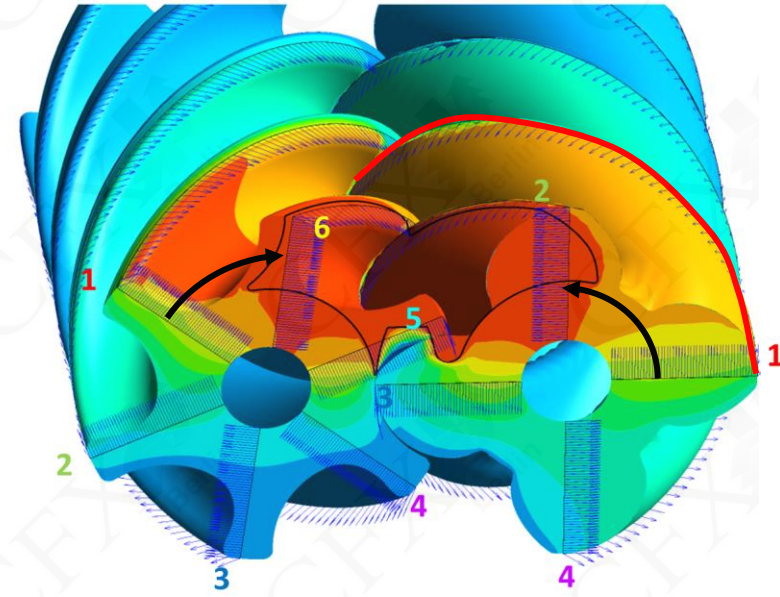
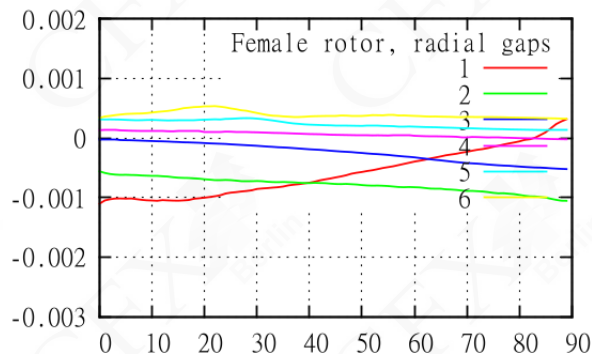
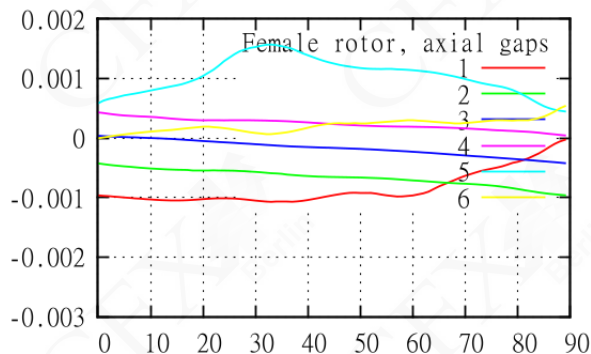
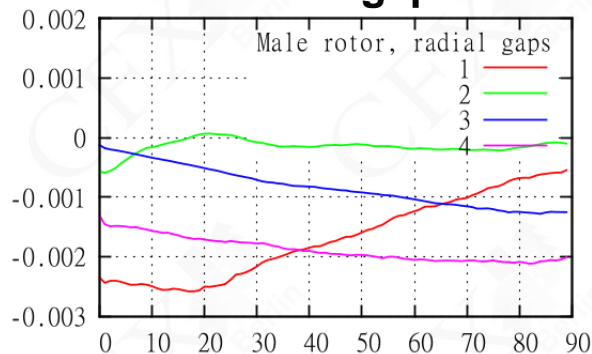
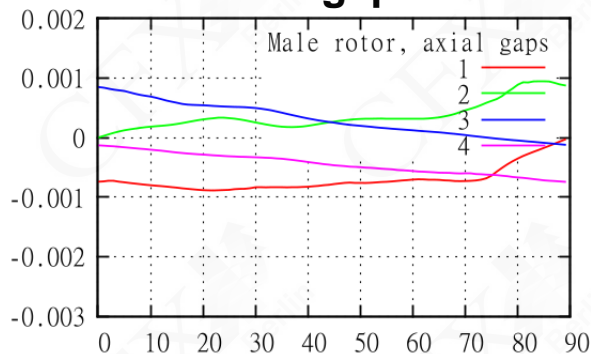
**Velocity vectors
in gaps**



- Evaluation of gap flow:
 - Generation of surfaces in axial and radial gaps allow evaluation of gap mass flow
 - Positive means: mass flow in rotation direction

Axial gaps

Radial gaps



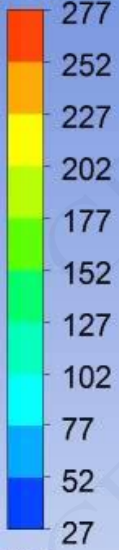
Male rotor

**Average mass flow
through compressor:
0.13 kg/s**

Female rotor

Temperature

Temperature



[C]

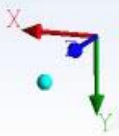
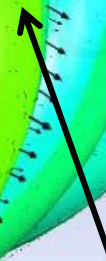
**Air temperature
on cross section**



0 deg

**Velocity vectors
in gaps**

**Solid temperature
on rotor surfaces**

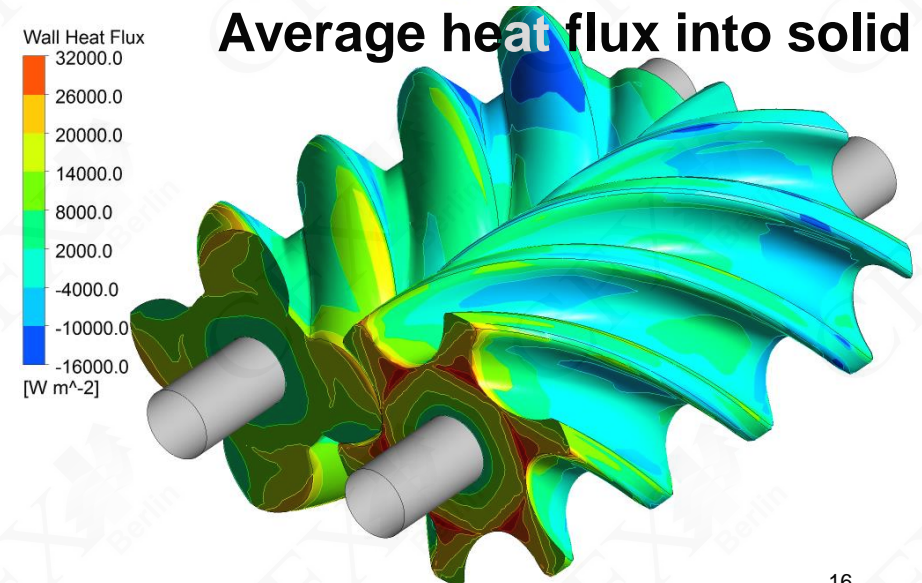
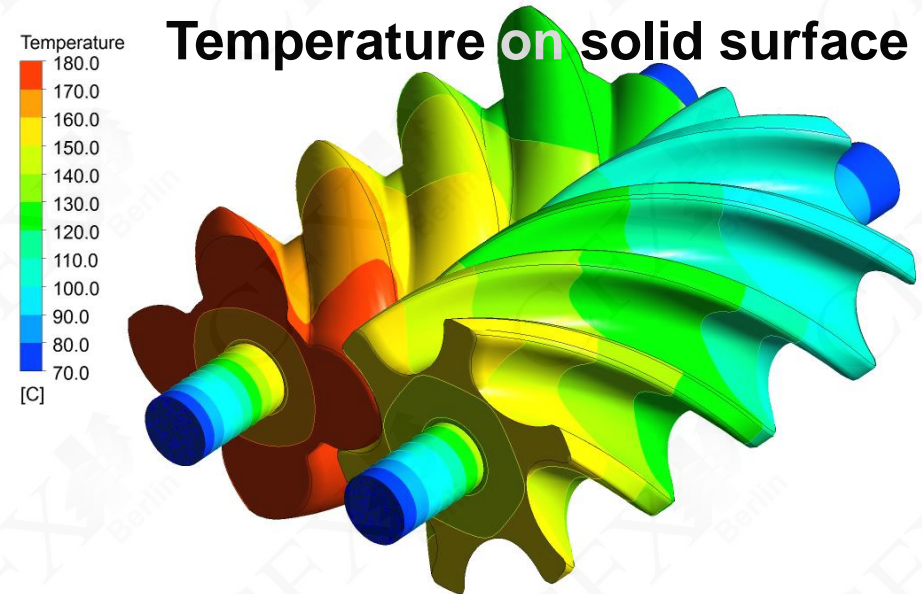


- **Heat exchange:**

- Compression of air increases air temperature
- Heat flux from hot air into cold solids heats solids
- Equilibrium typically after several minutes
- Simulation by iterative coupling of fluid/solid simulation (some ms) and heat load transfer to pure solid simulation (some min)

- **Equilibrium:**

- Male rotor heats up to 180°C, female up to 160°C due to 560 W heat flux at pressure side
- Cold gas at suction side is heated up with 380 W
- 180 W leave solid at shaft ends (fixed at 70°C)



- Efficient workflow from design to results
 - Design from basic literature and construction in Master's Thesis
 - High-quality meshes with TwinMesh and ANSYS Meshing
 - Setup, simulation, and post-processing with ANSYS CFX
 - Simulation of solid heat-up by iterative coupling of fluid/solid and pure solid simulation
 - Structural simulation of deformation by thermal and pressure loads
- Deep insight into complex physics of screw compressors
 - Visualisation of compression process by looking at pressures, velocities, temperatures, heat fluxes on surfaces, cross sections, etc.
 - Evaluation of compression process by looking at chamber pressures, mass flows, torque, power, or gap flows
 - Comparison of different designs or operating points
- TwinMesh and ANSYS CFD for
 - **Better and innovative designs** by a better understanding of complex phenomena
 - **Speed and flexibility at reduced costs** by massive use of virtual prototyping

Outlook

Oil Injected Screw Compressor

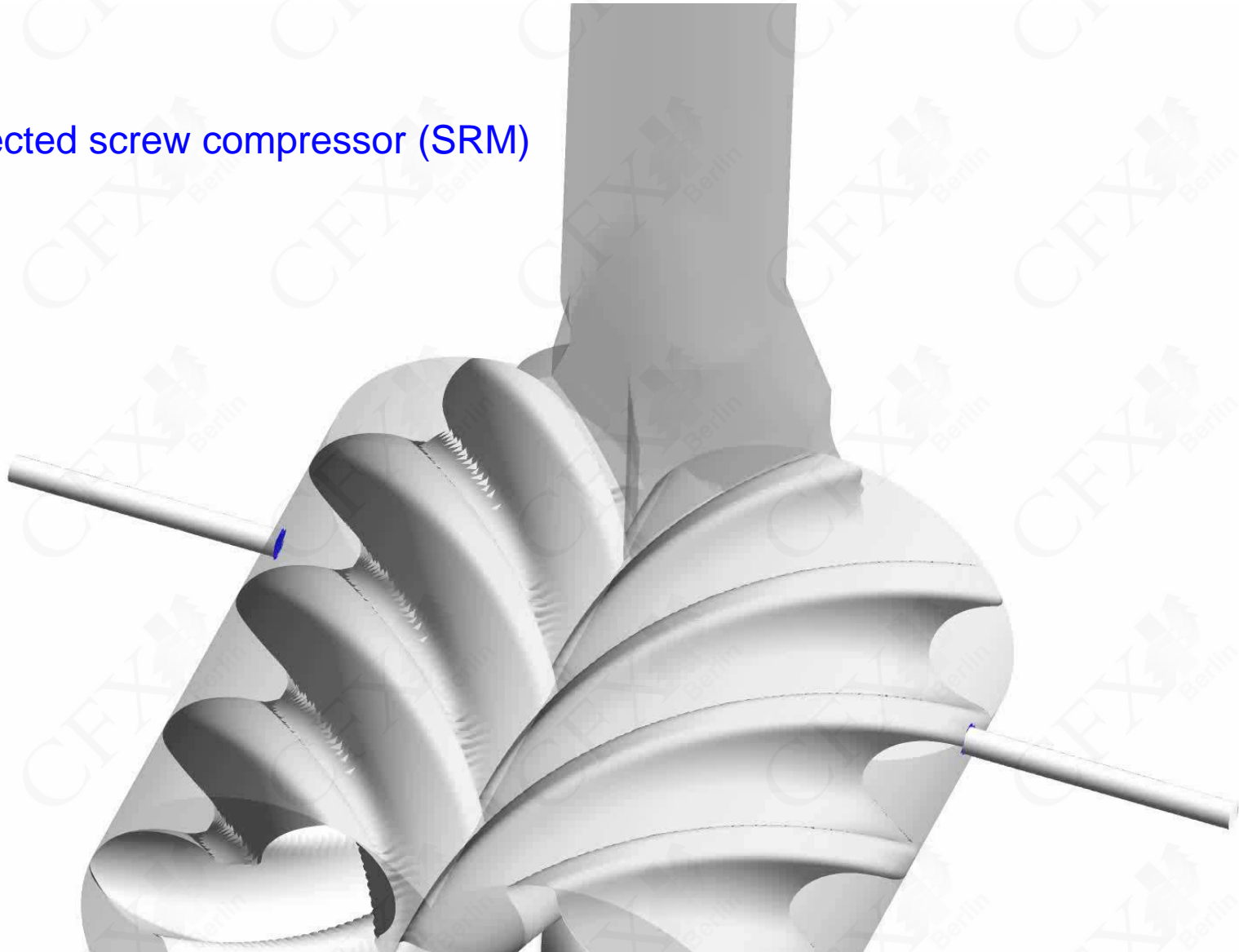
Oil.Temperature
75.00

Oil injected screw compressor (SRM)

50.00

25.00

[C]



Outlook

Screw Expander at 10 000 rpm

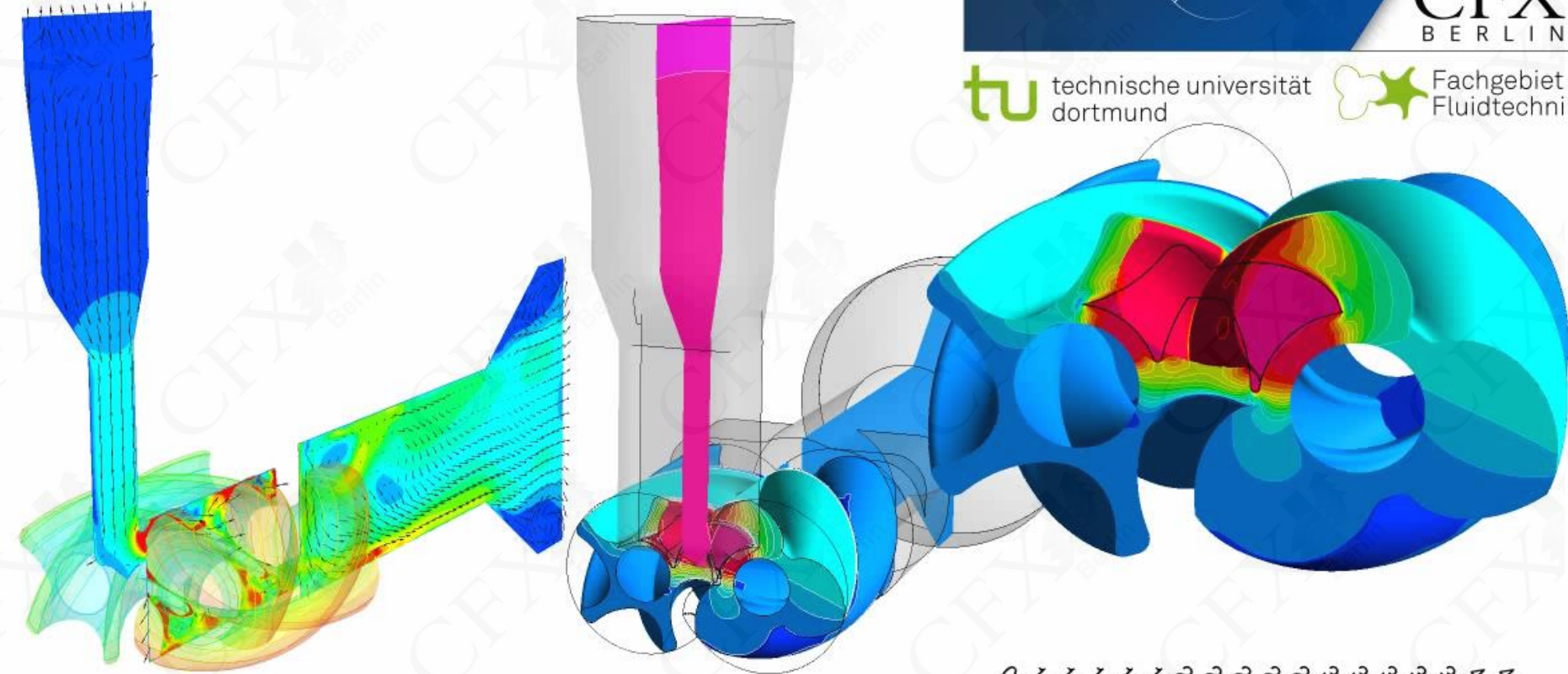
Velocity
40.0
36.0
32.0
28.0
24.0
20.0
16.0
12.0
8.0
4.0
0.0
[m s⁻¹]

Male rotation angle = 840 deg



tu technische universität
dortmund

Fachgebiet
Fluidtechnik

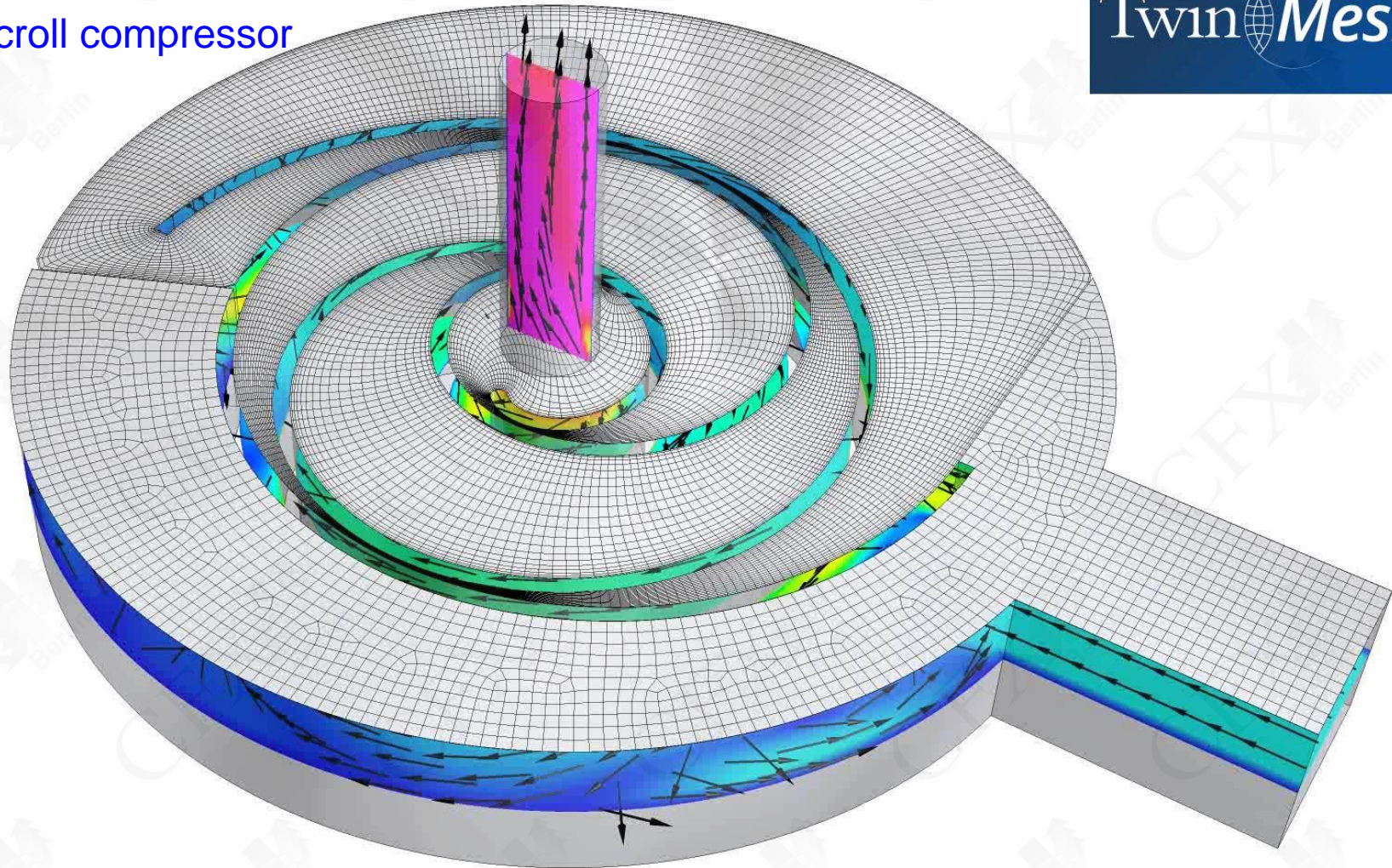


Screw expander at 10 000 rpm
(type GL-51 of TU Dortmund)

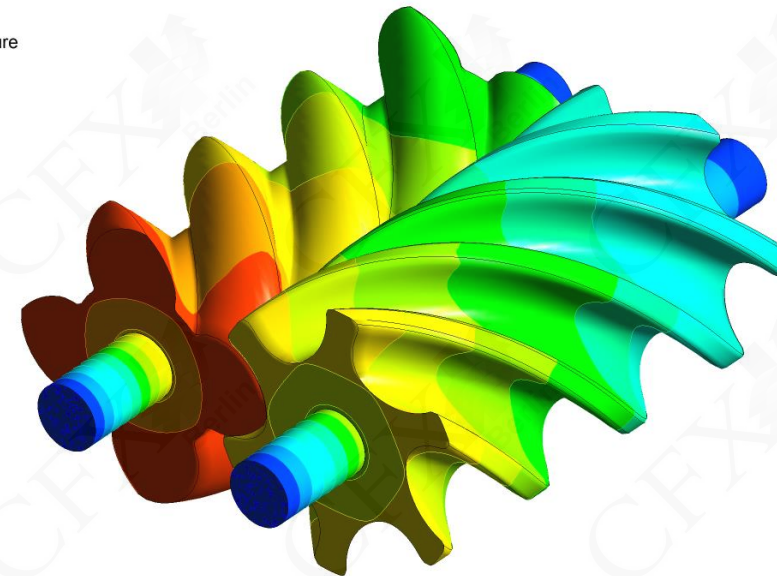
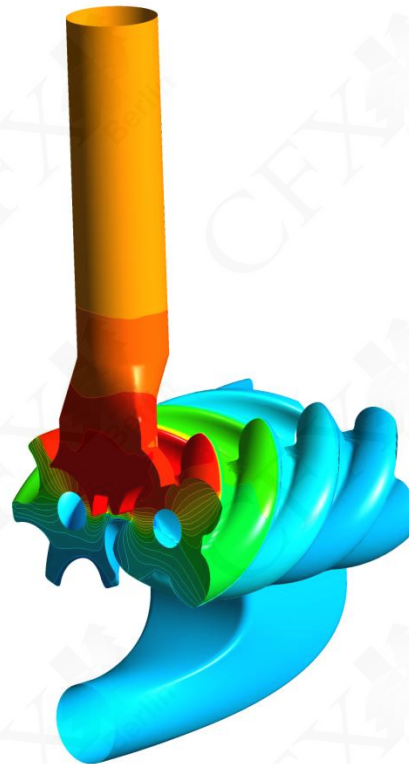
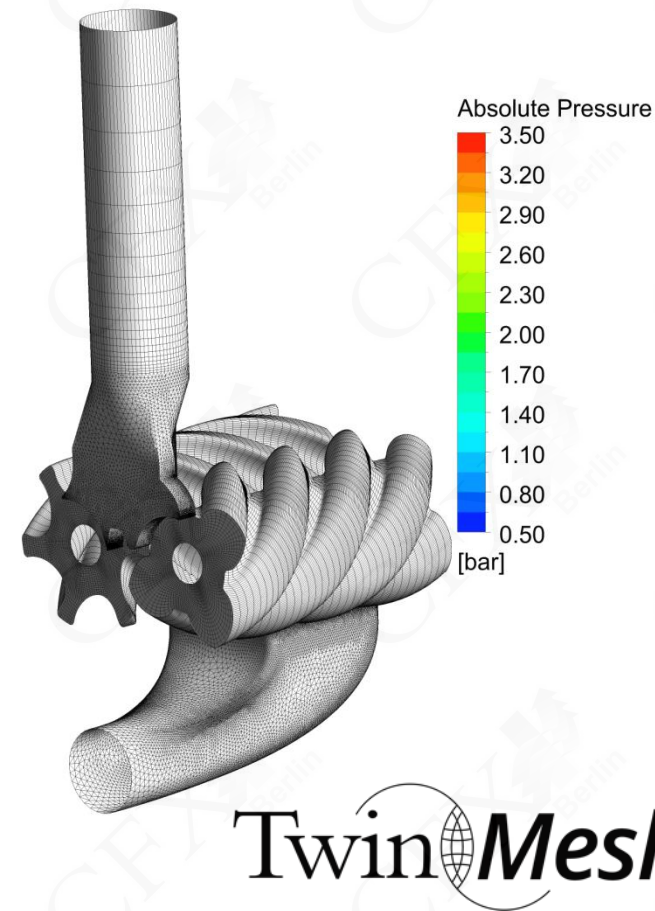
0.8 1.0 1.2 1.4 1.6 1.8 2.0 2.2 2.4 2.6 2.8 3.0 3.2 3.4 3.6 3.8 4.0 4.2 4.4 4.6 4.8 5.0 5.2 5.4 5.6 5.8 6.0 6.2 6.4 6.6 6.8 7.0 7.2 7.4 7.6 7.8 8.0 8.2 8.4 8.6 8.8 9.0 9.2 9.4 9.6 9.8 10.0 10.2 10.4 10.6 10.8 11.0 11.2 11.4 11.6 11.8 12.0 12.2 12.4 12.6 12.8 13.0 13.2 13.4 13.6 13.8 14.0 14.2 14.4 14.6 14.8 15.0 15.2 15.4 15.6 15.8 16.0 16.2 16.4 16.6 16.8 17.0 17.2 17.4 17.6 17.8 18.0 18.2 18.4 18.6 18.8 19.0 19.2 19.4 19.6 19.8 20.0
Absolute Pressure [bar]

Outlook Scroll Compressor

Scroll compressor



Thank you for your attention!



TwinMesh™

